Structurally Integrated Joint Torque Sensors in an Assistive Upper Limb Exoskeleton

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Abstract:
This paper describes the design process of a torque sensor, which is structurally integrated into the kinematic chain of an assistive upper limb exoskeleton to measure user interaction. Two different approaches of torque sensor integration are discussed regarding the nominal torque loads on each joint and applied transverse stresses. The final sensor concept consists of an aluminum spoke wheel deformation element with four strain gauges connected to a Wheatstone bridge. It is mounted between the motor casing and gear drive of each joint to measure the torque of the actuator and user. A sensor electronics board including signal processing and an EtherCAT interface is integrated in each joint. The torque sensor shows linear transmission characteristics for 0.4 Nm nominal rated torque with a linear error of 0.51 % and a hysteresis error of 0.53 %.

Key words: torque sensor, structurally integrated, assistive exoskeleton, user interaction, sensor electronics

Introduction
Ambient Assisted Living (AAL) comprises a set of methods, concepts and systems to support elderly and physically handicapped people to independently manage their activities of daily living [1]. The range of AAL solutions vary from software platforms to electromechanical systems, which assist users in complex movements of the upper and lower limbs. Here, an assistive upper limb exoskeleton (Fig. 1) is developed to provide haptic guidance during positioning tasks of the hand and arm. In order to recognize the movement intention or situation of the user without the need of additional bioelectrical signals, such as electromyography (EMG), torque sensors are integrated into the joints of the exoskeleton. Thus, derivations in the measured torque signals to the expected torque signals are interpreted as a user input.

Various approaches to recognize the user’s movement intention are applied in research and commercial products for assistive robotic systems. For exoskeletons, interaction force, torque, EMG measurements are used to generate desired movement trajectories for the user (Tab. 1). Since precise motion estimation based on EMG signals is difficult to realize [2], a system for user intention recognition based on structurally integrated torque sensors is proposed in this work.

Fig. 1: Assistive upper limb exoskeleton design with seven actuated degrees of freedom (DOF) q₁ ... q₇.
Several research groups have developed exoskeletons with movement intention recognition based on torque and force sensing [2-4]. Some approaches integrate an external signal processor to filter and preprocess the measured force and torque [3]. Some projects integrate torque or force sensors into the kinematic chain of the exoskeleton [5]. Less common are approaches using integrated sensor electronics [6, 7] with an analog-to-digital converter with 16-Bit and an FPGA to process the measured signals. For transmitting sensor and actuator data, a fieldbus communication system is implemented. Therefore, apart of the power supply, only one wire is necessary.

In the following sections, two different sensor design concepts are discussed and a sensor system with a sensor electronics board is evaluated. Finally, a concept for user intention recognition is introduced.

### Sensor Integration Concept

The most common method to measure the load signal on a rotation system is to transform a resulting surface strain with strain gauges, applied to a rotating shaft or deformation element, into an equivalent electrical signal. Since industrial torque sensors do not satisfy size and weight restrictions of the assistive robotic system, a structurally integrated sensor system is investigated. The main challenges in choosing the position of the sensing element in the joint structure of the exoskeleton are stiffness requirements of the kinematical chain. Concerning this, an approach with integrated strain gauges in the joint structure [11] is compared to a method with an additional deformation element [12].

The required nominal strain to measure with a foil strain gauge is 1 mm/m. Thus, the deformation element, integrated into the rigid links of the exoskeleton, must provide the required strain on the surface depending on the joint torque (Tab. 2).

### Tab. 1: Classification of upper limb exoskeletons

<table>
<thead>
<tr>
<th>Name, Reference</th>
<th>Joints</th>
<th>Application</th>
<th>DOF</th>
<th>Main control inputs</th>
</tr>
</thead>
<tbody>
<tr>
<td>[2]</td>
<td>Shoulder, elbow, wrist</td>
<td>Power assistance</td>
<td>7</td>
<td>sEMG, Torque/Force</td>
</tr>
<tr>
<td>[3]</td>
<td>Shoulder, elbow</td>
<td>Rehabilitation</td>
<td>4</td>
<td>Torque/Force</td>
</tr>
<tr>
<td>SAM, [4]</td>
<td>Shoulder, elbow, wrist</td>
<td>Power assistance</td>
<td>7</td>
<td>Torque/Force</td>
</tr>
<tr>
<td>SUEFUL-7, [8]</td>
<td>Shoulder, elbow, wrist</td>
<td>Power assistance</td>
<td>7</td>
<td>sEMG, Torque/Force</td>
</tr>
<tr>
<td>[9]</td>
<td>wrist</td>
<td>Power assistance</td>
<td>1</td>
<td>EMG, EEG</td>
</tr>
<tr>
<td>[10]</td>
<td>Shoulder, elbow, wrist</td>
<td>Rehabilitation</td>
<td>9</td>
<td>EMG, Force</td>
</tr>
</tbody>
</table>

### Tab. 2: Computed maximal joint torques with 20 N load at the end effector.

<table>
<thead>
<tr>
<th>Joint</th>
<th>q1</th>
<th>q2</th>
<th>q3</th>
<th>q4</th>
<th>q5</th>
<th>q6</th>
<th>q7</th>
</tr>
</thead>
<tbody>
<tr>
<td>Torque (Nm)</td>
<td>29</td>
<td>26</td>
<td>21</td>
<td>15</td>
<td>9</td>
<td>5</td>
<td>2</td>
</tr>
</tbody>
</table>

The rigid links of the exoskeleton consist of a combination of carbon fiber reinforced polymer (CFRP) and aluminum alloy. The mechanical power of the BLDC motors is transitioned via Harmonic Drive® precision gears (gear ratio 1:100, efficiency 0.85) at each joint. Stress simulations of the link elements show that the required strain on the surface is obtained by weakening the material at certain positions (Fig. 2, left). Since the resulting strain appears on both sides of the element, it is possible to measure the joint torque with a half-bridge configuration (Fig. 2, right). The advantages of this approach is a direct measurement of the joint torque at the link element of the exoskeleton, no extra deformation element is necessary.

![Fig. 2: FEM stress simulation of the joint link with nominal joint torque (left). Concept for torque sensor integration with strain gauges into kinematic structure of the exoskeleton joint axis (right).](image_url)

Since the mechanical properties of CFRP are depended on the structure of the composite material (mostly unidirectional carbon filament), the E-modulus and strain transmission behavior is analyzed in a four-point bending test rig (Fig. 3).
The measurement results show an E-modulus of 104 GPa and linear transmission behavior (linearity error < 1 %) of the CFRP probe. The weight of the exoskeleton generates a torque perpendicular to the rotational joint axis, which applies transverse stress to the sensing element. The first two shoulder joints suffer from the highest transverse stress (resulting torque \( \tau_{s}(q_1) = 28 \text{ Nm}, \tau_{s}(q_2) = 10.5 \text{ Nm} \)). Since the structure of the rigid body is weakened for the strain transmission of the sensing element, the transverse stress exceeds the tensile strength of the CFRP part. Thus, direct torque measurement at the joint link with foil strain gauges risks damage to the exoskeleton parts.

As an alternative, it is possible to measure the torque between the actuator and gear drive. This leads to a reduction of the load to the sensor in proportion to the gear ratio (1/100) and a decoupling of the sensing element to the kinematic structure. In order to reduce weight and save space, the sensing element is integrated into the flange of the motor (Fig 4.)

**Sensor Design and Calibration**

The geometry of the deformation element is based on a spoke wheel structure to transduce the motor torque into a strain that is measured with four strain gauges. The sensing element is used as a flange to connect the motor with the gear drive. The dimensions of the spoke wheel bridges are designed depending on the applied torques by the motors. The nominal torque of the BLDC motors for the first and second DOF of the shoulder joint is 0.379 Nm. With FEM simulations, the deformation element is dimensioned with a bridge size of 4 x 0.45 mm (Fig. 5, right).

![Deformation element geometry with strain gauge positions (left) and spoke wheel dimensioning with FEM (right).](image)

The deformation element is CNC manufactured using aluminum alloy, polished and cleaned. The dimensions of the bridges allow a simple positioning of the strain gauges, using the edges as landmarks. Four strain gauges (type 1-LA11K3/350-E, Hottinger Baldwin Messtechnik GmbH, Germany) are applied with cyanoacrylate adhesive (type Z70, HBM) and connected to a full bridge circuit. The sensor is calibrated in both directions with weights via a defined lever arm (Fig. 6, left). At first, the sensor is loaded three times with the maximum weight (0.35 Nm). Secondly, the sensor is loaded and unloaded with 50 g weights in nine steps (Fig. 6, right). The voltage is measured with a 6-wire bridge amplifier (type EL3356, Beckhoff Automation GmbH & Co. KG, Germany).

![Torque sensor calibration setup with weights (left) and calibration curve of sensor "shoulder 2" (right).](image)

The sensor shows a linear transmission behavior with only small hysteresis error (Tab. 3).

**Tab. 3: Sensor calibration results.**

<table>
<thead>
<tr>
<th>Sensor</th>
<th>Gain (mV/Nm)</th>
<th>Lin. error</th>
<th>Hys. error</th>
</tr>
</thead>
<tbody>
<tr>
<td>Shoulder 1</td>
<td>11800</td>
<td>0.012</td>
<td>0.021</td>
</tr>
<tr>
<td>Shoulder 2</td>
<td>12300</td>
<td>0.005</td>
<td>0.005</td>
</tr>
</tbody>
</table>

A reason for the occurring measurement error is a position error due to manual application of the strain gauges.
Sensor Electronics and System Evaluation

Strain gauges change their resistance when they are stretched or compressed. The change of resistance is small and requires sensitive circuiting to measure accurately. The Wheatstone bridge is a common circuit to convert the small resistance change into a measurable voltage. Depending on the number of used strain gauges there are different possibilities to wire up a Wheatstone bridge. For the integrated torque sensor, a full-bridge is chosen, providing a high sensitivity and temperature compensation. In contrast to a current source for the excitation supply, using a voltage supply needs no additional hardware. Additionally, the lead resistance is negligible with four-terminal sensing.

The Wheatstone bridge provides a full-scale of 2mV/V with 350 Ω strain gauges. With an excitation supply of 5V, an output of 10 mV is generated. To exploit the scale range of an analog-to-digital converter (ADC), an amplifier is essential. The bridge excitation is used simultaneously as the reference voltage for the analog-to-digital converter (Fig. 7). As a result, any drift in excitation voltage also influences the reference voltage and removes the drift error.

For integrated measurement data analysis, a ΔΣ analog-to-digital converter (type ADS1262, Texas Instrument, USA) is used with an integrated PGA with an amplification up to 32, a resolution of 32-bit and a sampling rate up to 38.4 kSPS. The signal is processed with a microcontroller, providing an EtherCAT interface for signal integration into the global fieldbus of exoskeleton. The complete sensor electronics board is integrated into each joint of the exoskeleton. Apart of the Ethernet connection and wires for the energy supply, no additional wires are necessary. To reduce noise, the length of the lead wire is as short as possible. The leads of the signal, the excitation and the remote sense are twisted. The lead resistance is negligible since the four-terminal sensing is applied.

The weight force of the motor leads to an error (~ 3 % of the nominal torque), depending on the joint angles of the exoskeleton. Since the gravity vector is measured through an IMU at the base of the exoskeleton, the error of the motor weight is cancelled out with signal processing.

The designed sensor board is connected to a motor control stack (type IFM Drive DC 300, Core C22, COM EtherCAT, Synapticon, Germany) via SPI and placed on the rigid link next to the joint of the exoskeleton (Fig. 8).

When driving the motor with a position controlled sinus joint angle, a repeatable stick-slip effect is noticeable (Fig. 9). The error is caused by friction in the gear drive and must be considered in joint torque computations.

Fig. 7: Measurement chain with signal amplification, ADC and digital signal processing.

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Fig. 8: 2 DOF shoulder exoskeleton with integrated torque sensors and electronics.

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Therefore, the desired torque of the exoskeleton \( \tau = \tau_G(q) + \tau_I(q) + \tau_F(q, \dot{q}, \ddot{q}) + \tau_EF + \tau_U \) is the sum of torques caused by gravity \( \tau_G \), inertia \( \tau_I \), friction \( \tau_F \) [13], the end effector (EE) interaction \( \tau_EF \) [14] and the user interaction \( \tau_U \).

If the user is not applying any torque to the system, the torque sensor only measures the dynamics and interaction with the environment (Fig. 11, blue curve). If the user is applying a torque to the joints of the exoskeleton during the same operation, the measured torque increases (Fig. 11, green curve).

**Conclusion**

Measuring joint torque in an exoskeleton is a challenging task. Torque measurement in the joints of the kinematic chain is not possible with regular foil strain gauges due to high transverse forces. Therefore, the sensing element is integrated between the motor and gear of each joint in order to reduce the influence of shear forces and torques. The designed sensor shows excellent transmission behavior for torque measurements in the robotic joints. Thus, movement intentions, applied through joint torques by the user, are detected with the sensor system without using motor current or EMG.

**Outlook**

The measurement results show a strong impact of friction in the gear drive to the sensor signal. In the future, a dynamic model for friction in the gear drive will be created and evaluated with measurement data from different gear drives.

**References**


